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PM Emissions from HCCI Engines

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Jaguar Cars

**Particulate matter
and measurement**

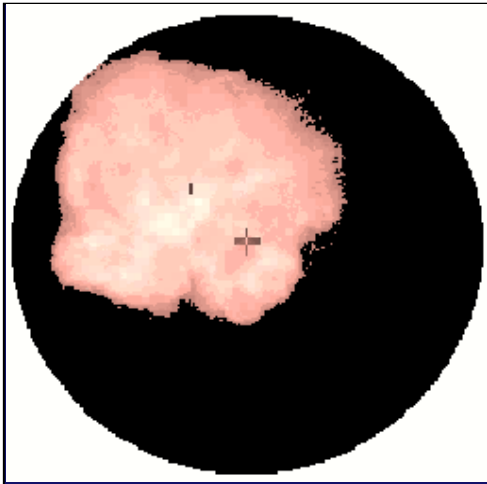
Cambridge University, 16 May 2008



Presentation outline

- **Background**
- **PM with Gasoline HCCI**
 - Varied valve timing
 - Varied injection timing
 - Effect of air temperature
 - Comparison with SI combustion
 - Effect of diesel addition
- **Summary and conclusions**

HCCI – a generic name for new combustion mode

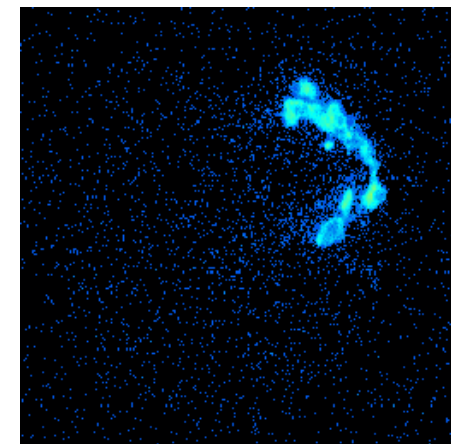


SI combustion



Diesel combustion

(Oxford Lasers)



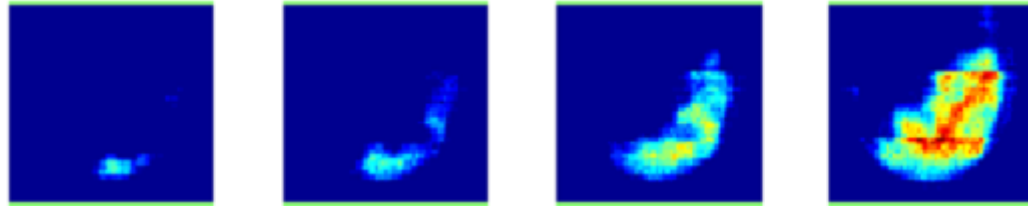
Homogeneous Charge
Compression Ignition

- **spontaneous multiple-point ignition**
- **typically, rather homogeneous mixture and temperature distributions**
- **capable of running with extremely lean mixtures (AFR>80)**
- **load can be controlled by fuel quantity or air-EGR dilution**

Advantages of HCCI/CAI combustion

- Extremely low (1-5% of SI) NO_x emissions
 - absence of high temperature regions
- High fuel economy (approach Diesel's)
 - un-throttling, lean combustion possible, increased heat release rate, removal of knock tendency, high CR possible
- Very low cycle-by-cycle variations ?
 - multipoint flame initiation

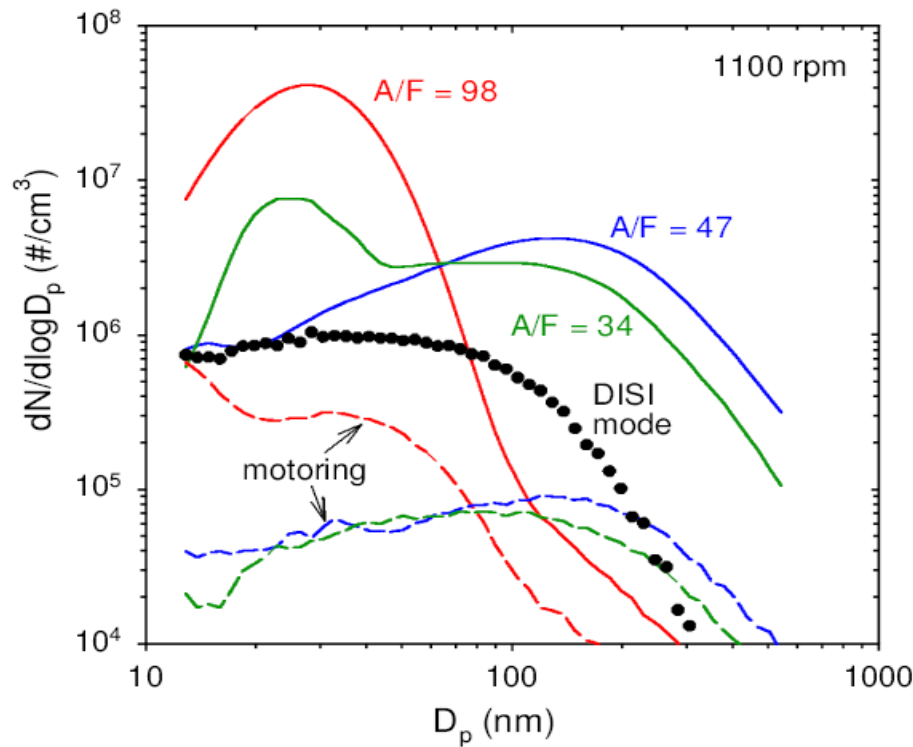
- **“Smokeless” ?**



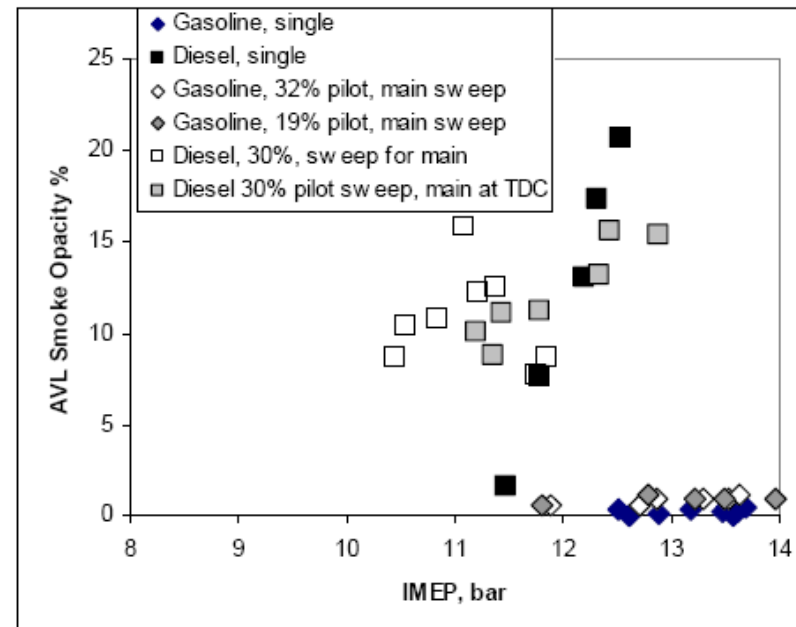
Combustion images – Wilson *et al* SAE Paper 2005-01-2129

How much PM is HCCI producing

The first measurement of HCCI PM by Ford was 'surprising', although clearly HCCI smoke numbers will be low compared to diesel

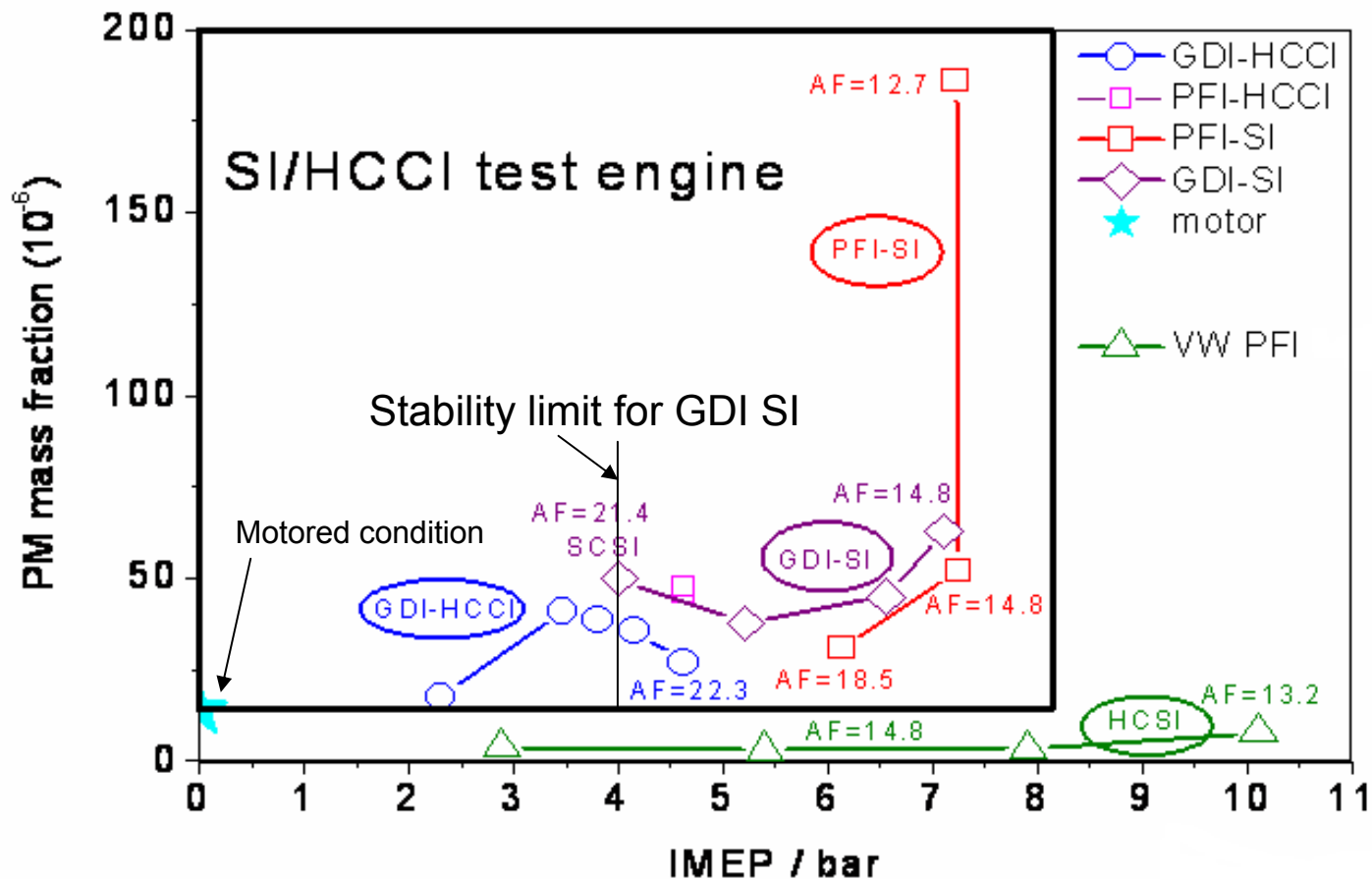


Kaiser *et al*, 2002



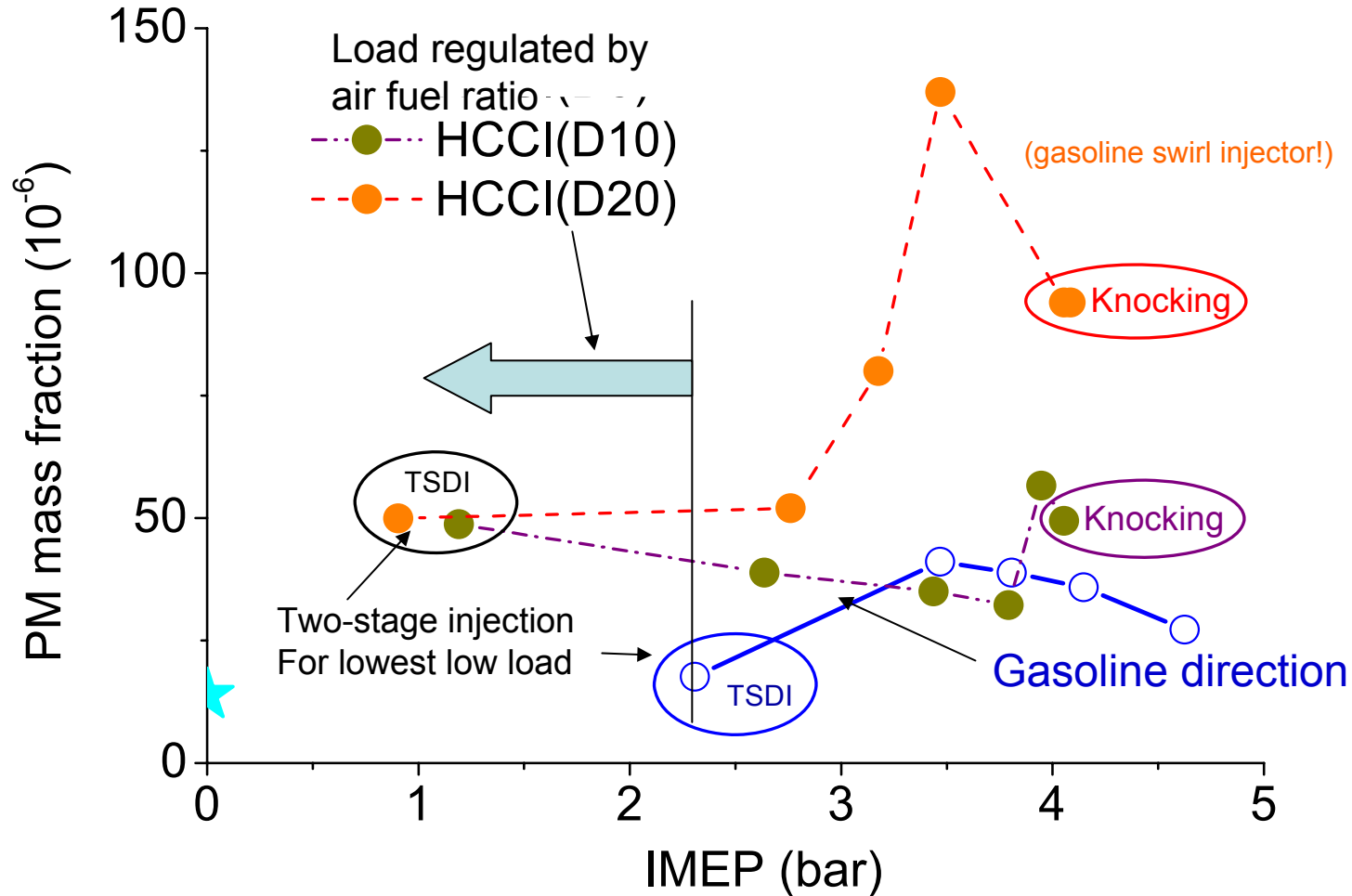
Kalghatgi *et al*, 2007

Comparison of PM for different combustion systems



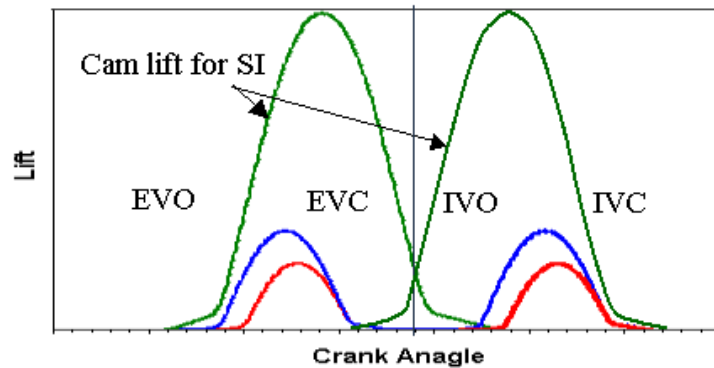
The load of the GDI engine is regulated by air fuel ratio and stratified charge is used for low load - **Data measured at Tsinghua University**

Extension of the low load HCCI boundary by **Dieseline**



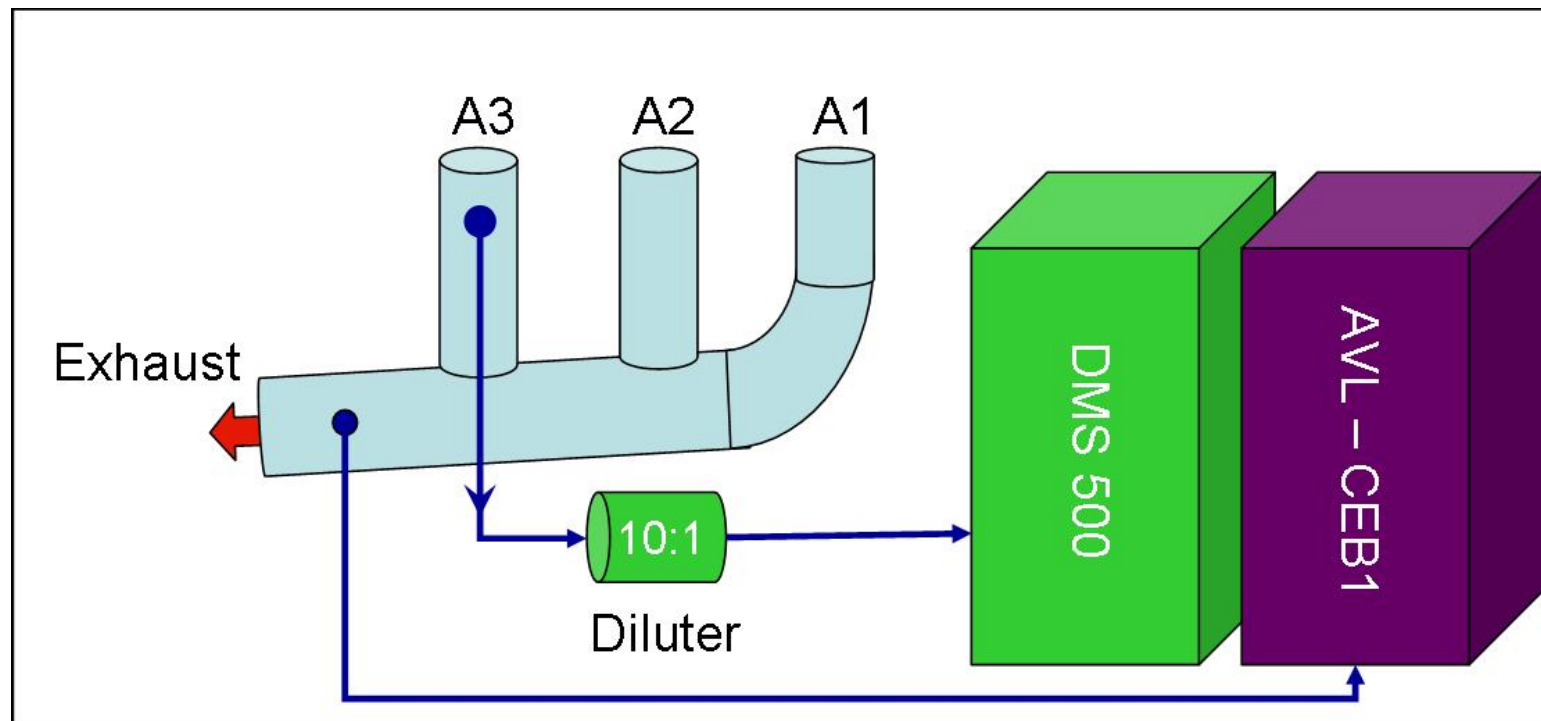
Dieseline- Mixture of gasoline and diesel fuels - Data measured at Tsinghua University
Xu *et al*, 2008 SAE HCCI Symposium

Jaguar HCCI engine

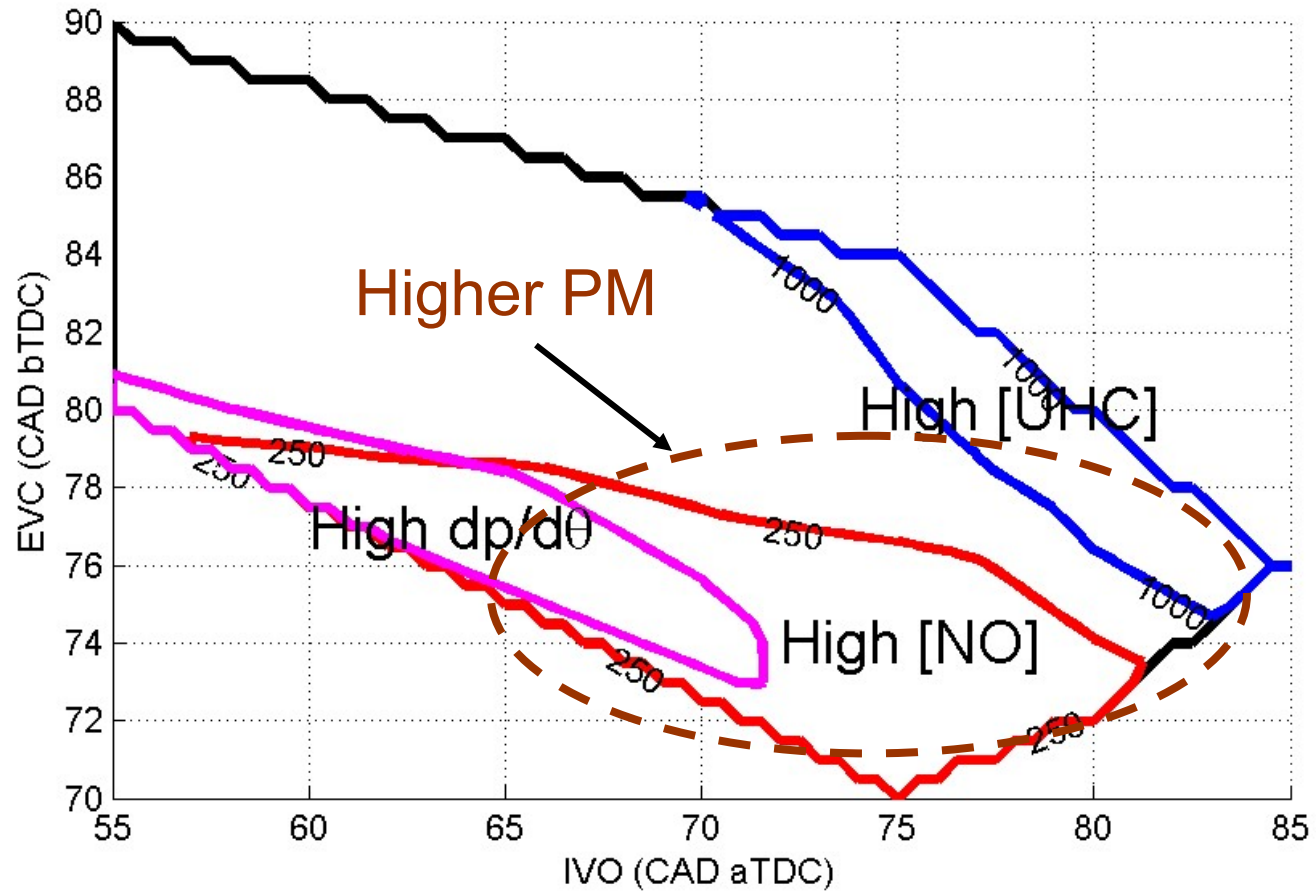


Base Engine	3.0L V6
Bore	89 mm
Stroke	79.5 mm
Compression Ratio	11.3
Con Rod Length	138.1 mm
Fuel	UL95 gasoline
Dual Variable Cam Timing (VCT)	
Dual Cam Profile Switching (CPS)	
Direction injection	

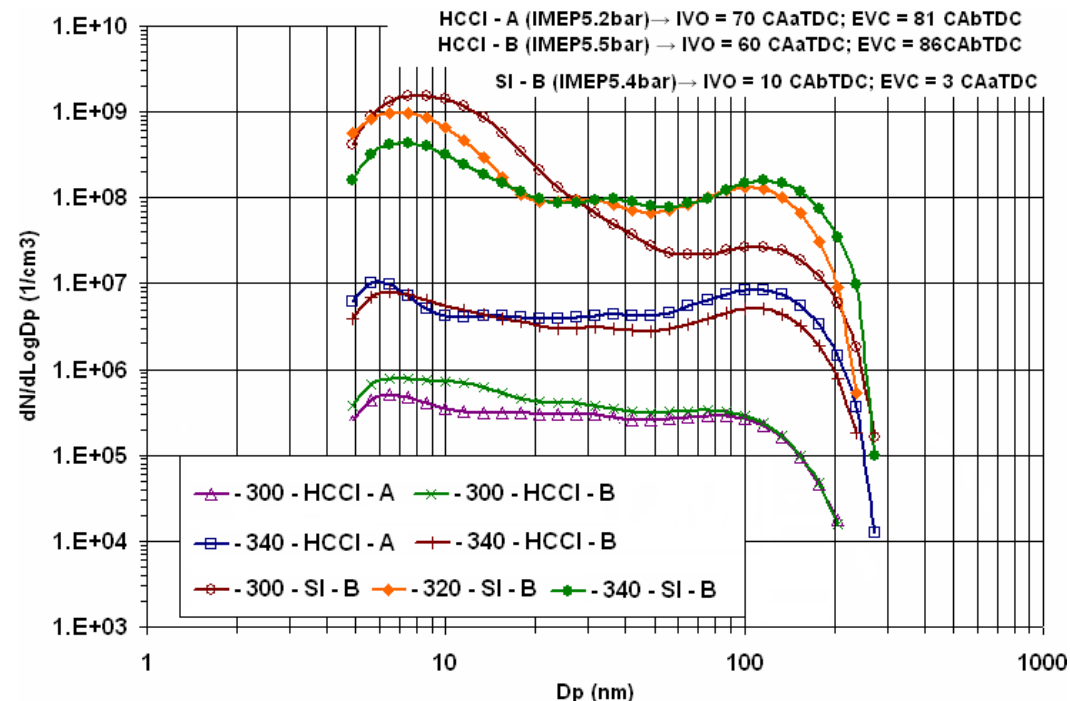
Emission measurement system



Emission characteristics of the Jaguar HCCI engine



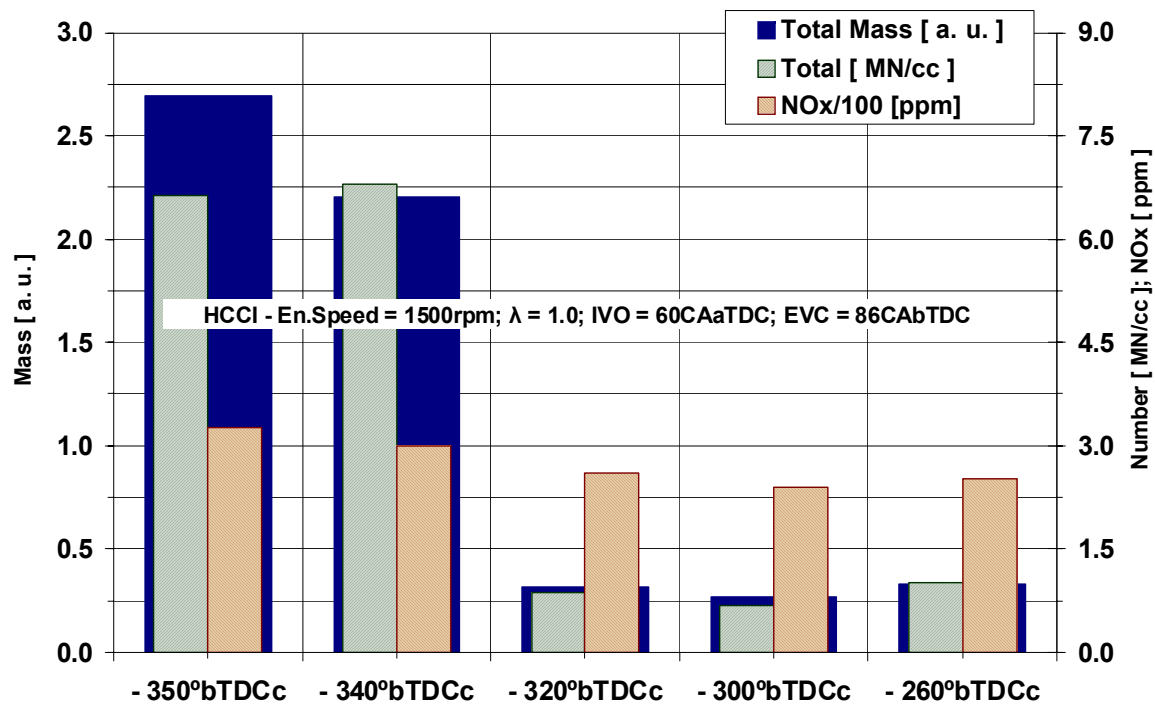
PM comparison for HCCI and SI



1. At a higher load condition for IMEP of 5.2-5.4bar, PM with HCCI shows a lower level in both accumulation mode and nucleation mode than SI
2. Varied injection timing (-300 or -340) can make a big difference, greater for that in the case of varied valve strategies (A or B) or SI

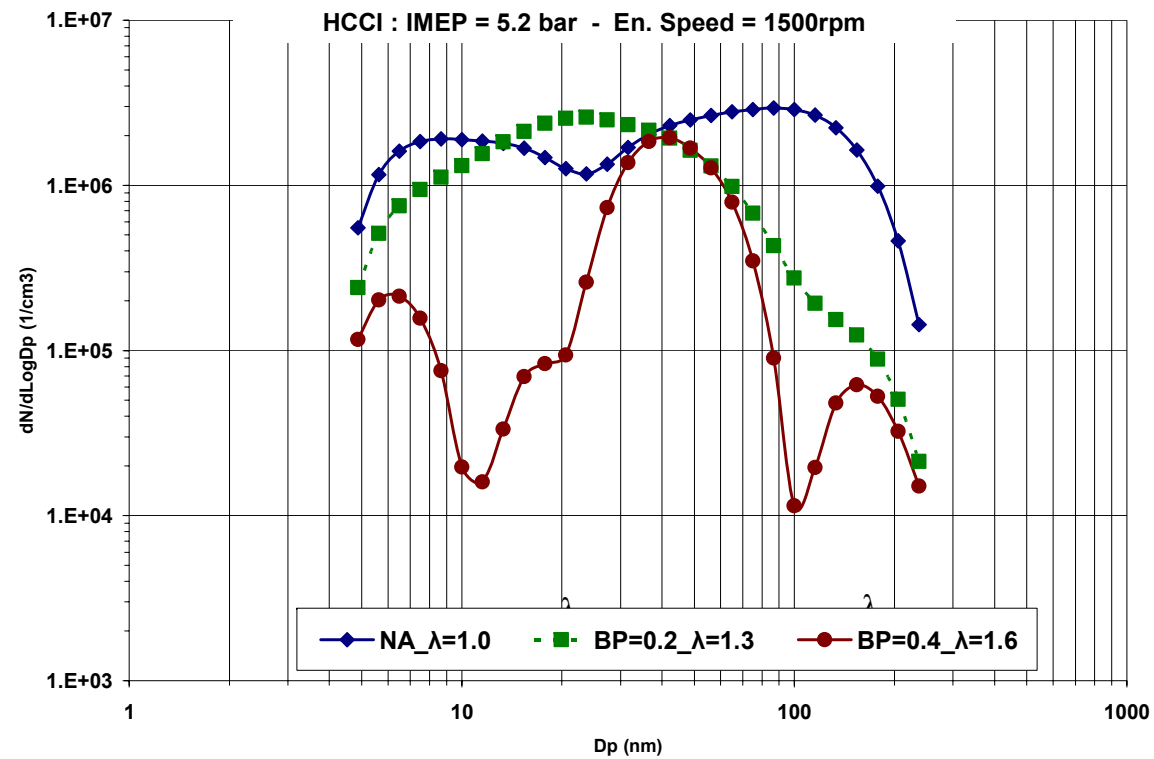
PM for varied injection timings - HCCI

HCCI - 1500rpm; $\lambda = 1.0$; IVO = 60CAaTDC; EVC = 86CAbTDC



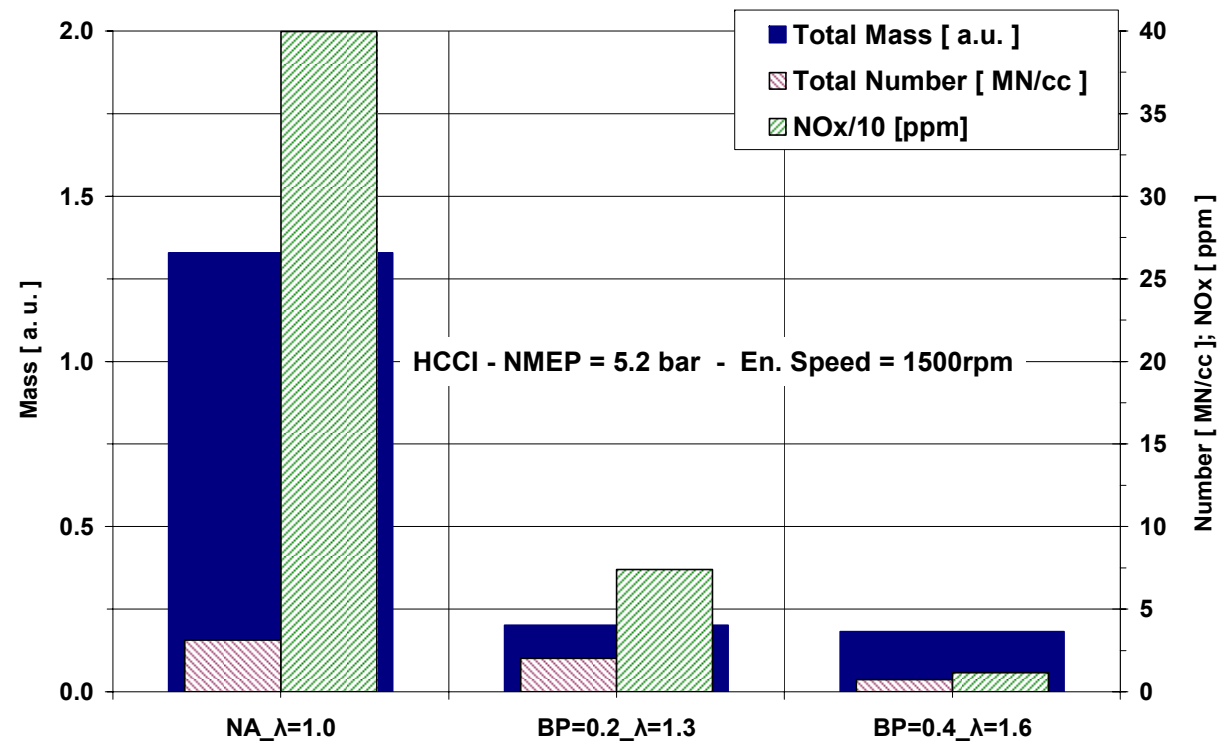
Matching of the injection and the intake valve timing may be important

PM for lean burn with boosting - HCCI



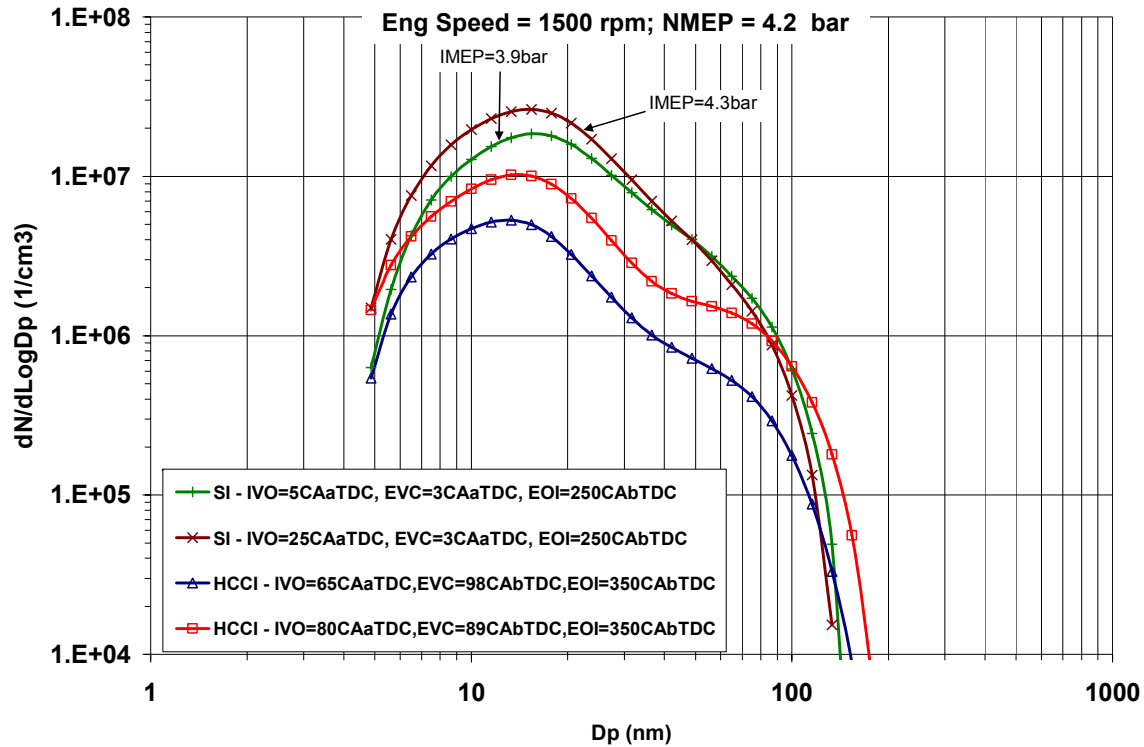
lean burn significantly reduces PM
over the full spectrum

PM for lean burn with boosting - HCCI



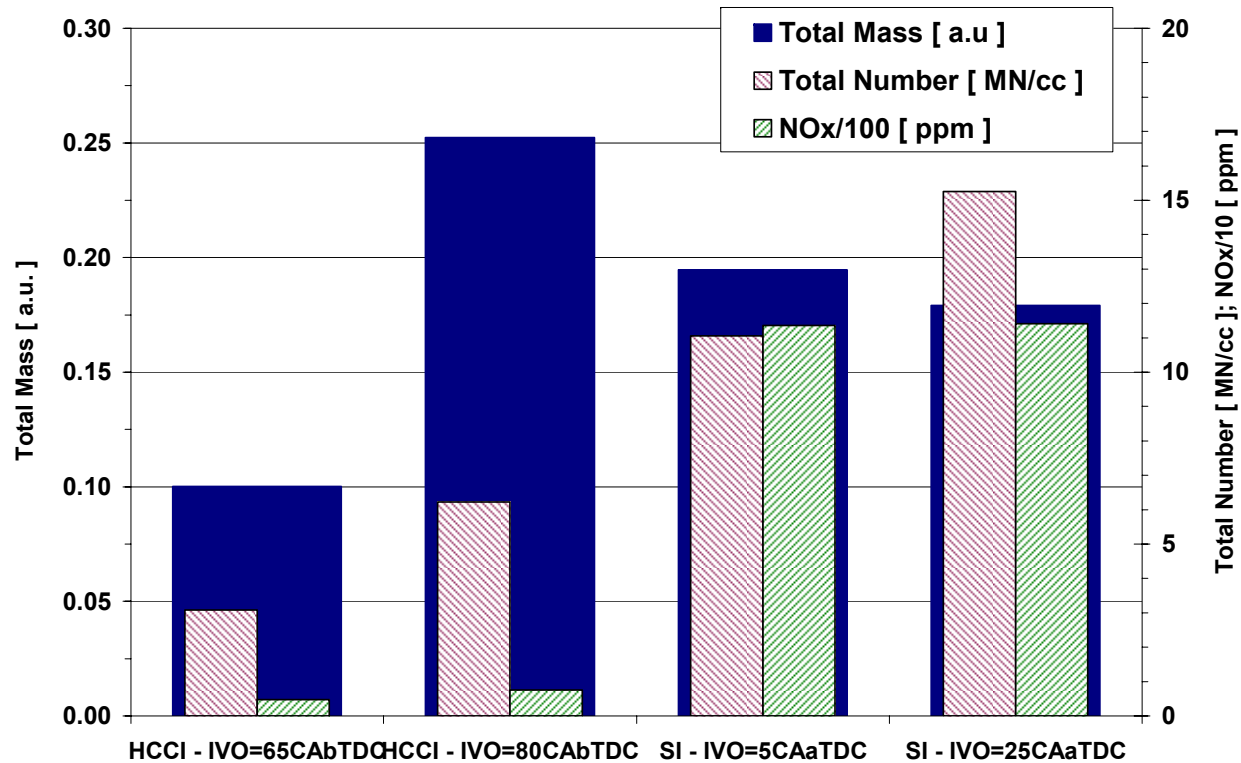
lean burn significantly reduces PM, both in mass and number

PM may approach SI for smaller load



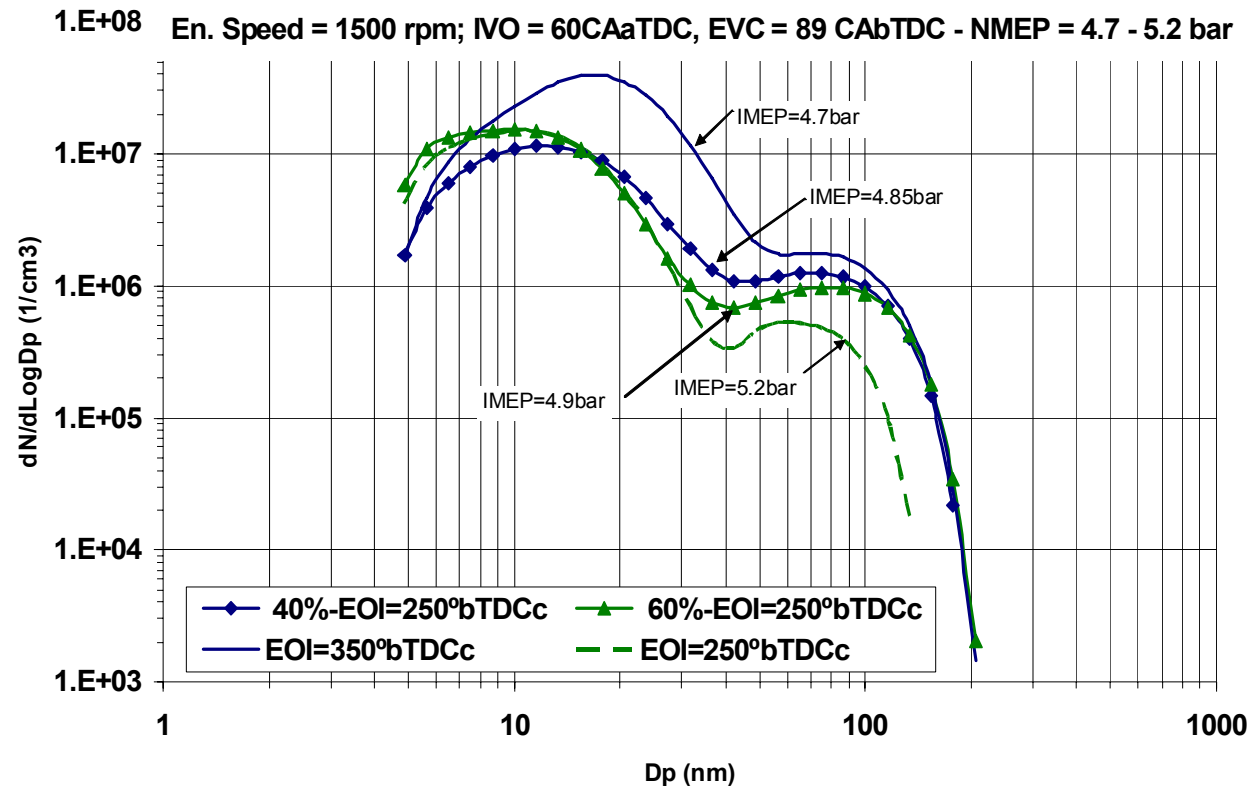
There are relatively more larger size of PMs with HCCI at lower load (increase of accumulation mode fractions).

PM may approach SI for smaller load



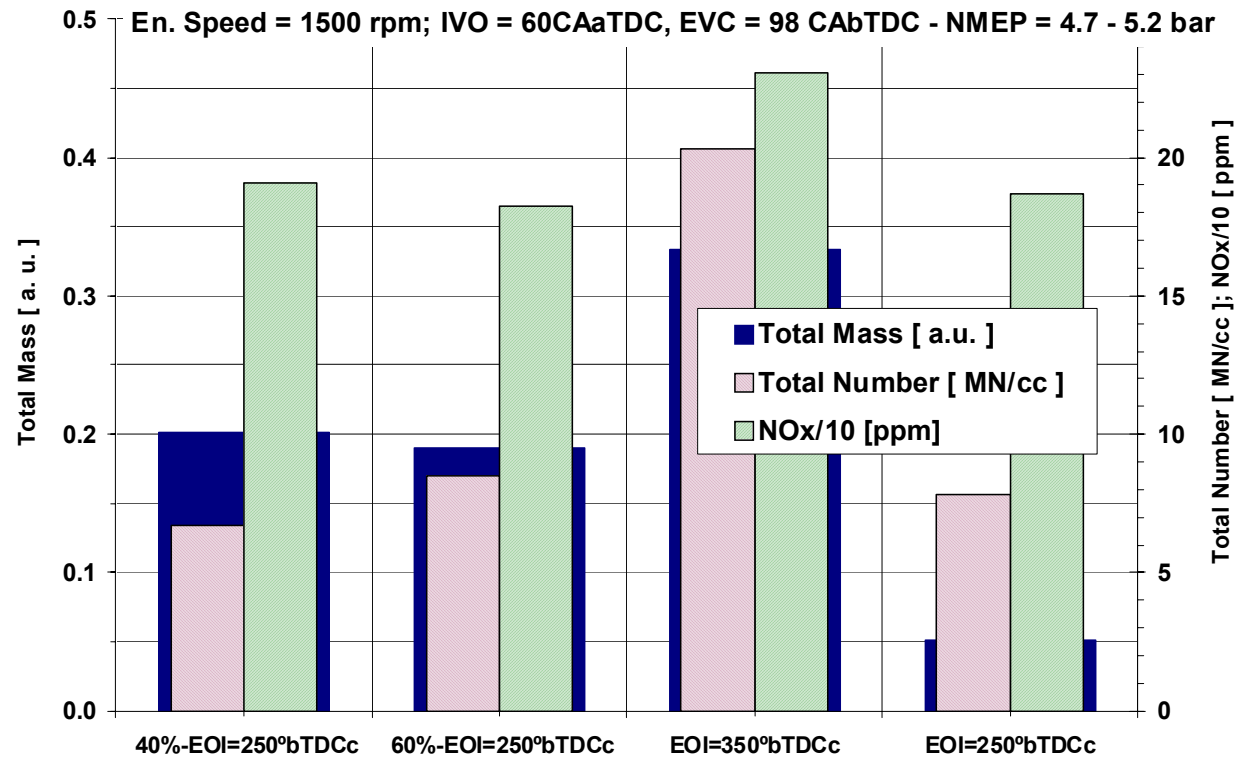
In one case, PM mass even exceeded SI case for the same load

PM with split injection



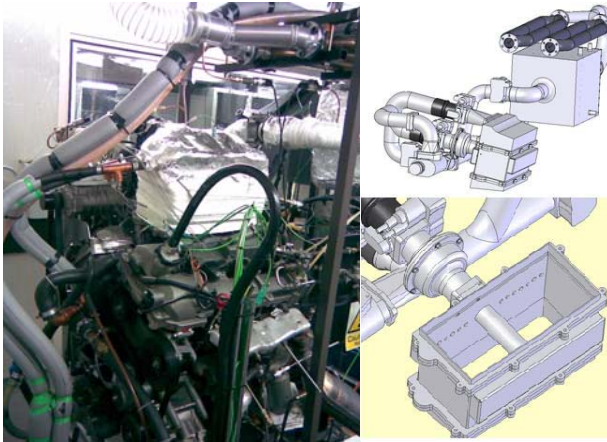
PM will fall between the boundaries for one injection in recompression or late induction

PM with split injection



More pilot injection may increase PM

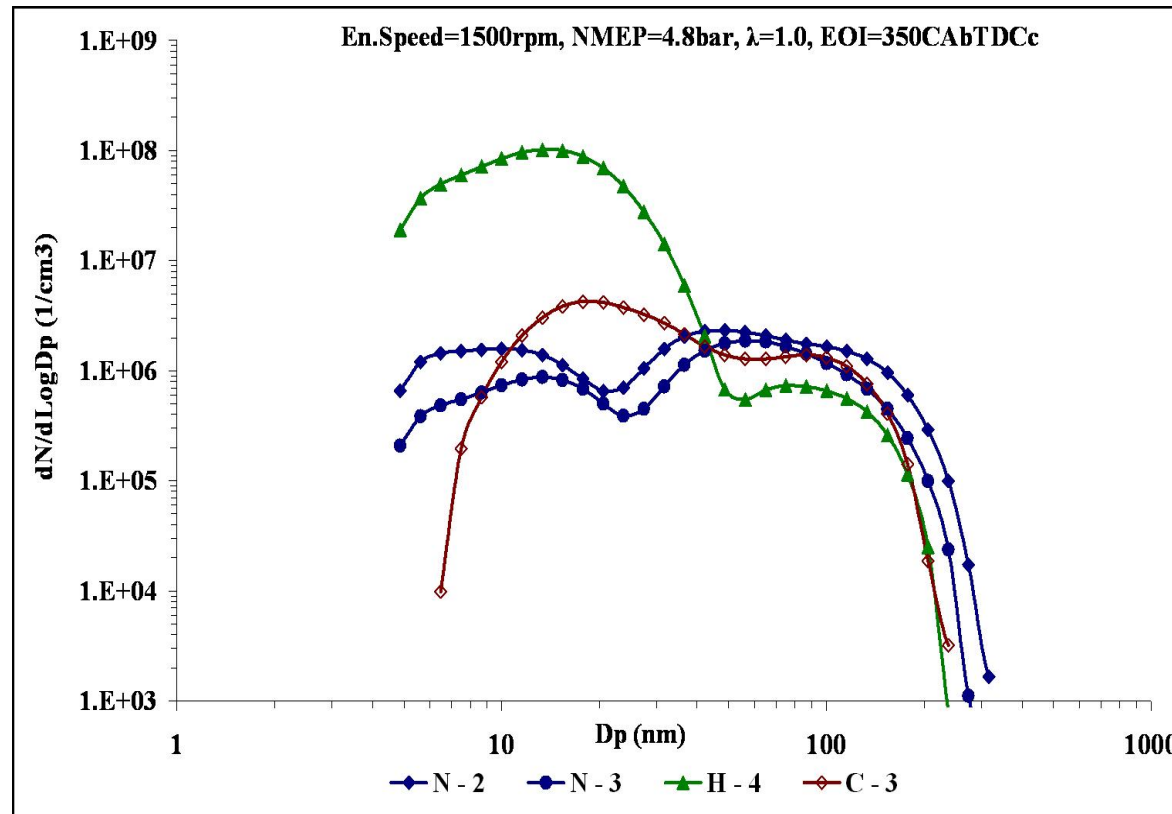
Effect of intake air temperature



A thermal management system incorporating electrical heaters and an intercooler bypass was used to control the intake air temperature. Varied valve timings are used to regulate the engine load

	TEST	NMEP	dp/Θ	IVO	EVC	T_In	T_Ex	PM No. 50-300nm	PM Mass 50-300nm	EGR	η _{vol}	η _{comb}
		bar	bar/deg.	°aTDC	°bTDC	° C	° C	MN/cc	a. u.	%	%	%
Ambient Charge Air	N - 2	4.8	6.0	60	86	56	412	0.90	0.69	41.3	60.9	93.6
Ambient Charge Air	N - 3	4.8	4.2	80	81	61	430	0.65	0.36	40.5	60.3	93.5
Heated Charge Air	H - 4	4.8	5.8	60	91	87	391	0.29	0.18	44.3	53.1	93.6
Cooled Charge Air	C - 3	4.8	2.4	80	86	45	438	0.57	0.31	50.3	56.0	92.7

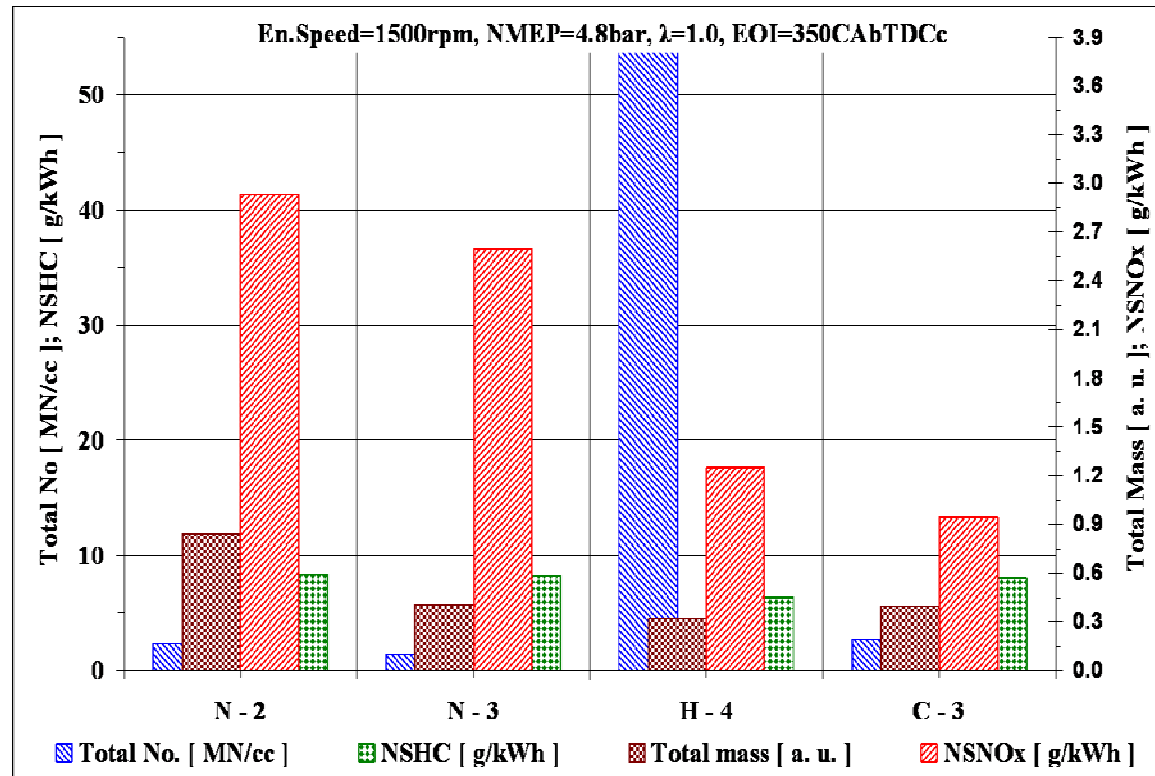
Effect of intake air temperature (1)



N – normal, H – heated, C – cooled inlet air

Higher intake temperature seems to increase the number of PM in nuclei mode

Effect of intake air temperature (2)



The effect of intake temperature on the mass of PM is limited, but apparently more on the number

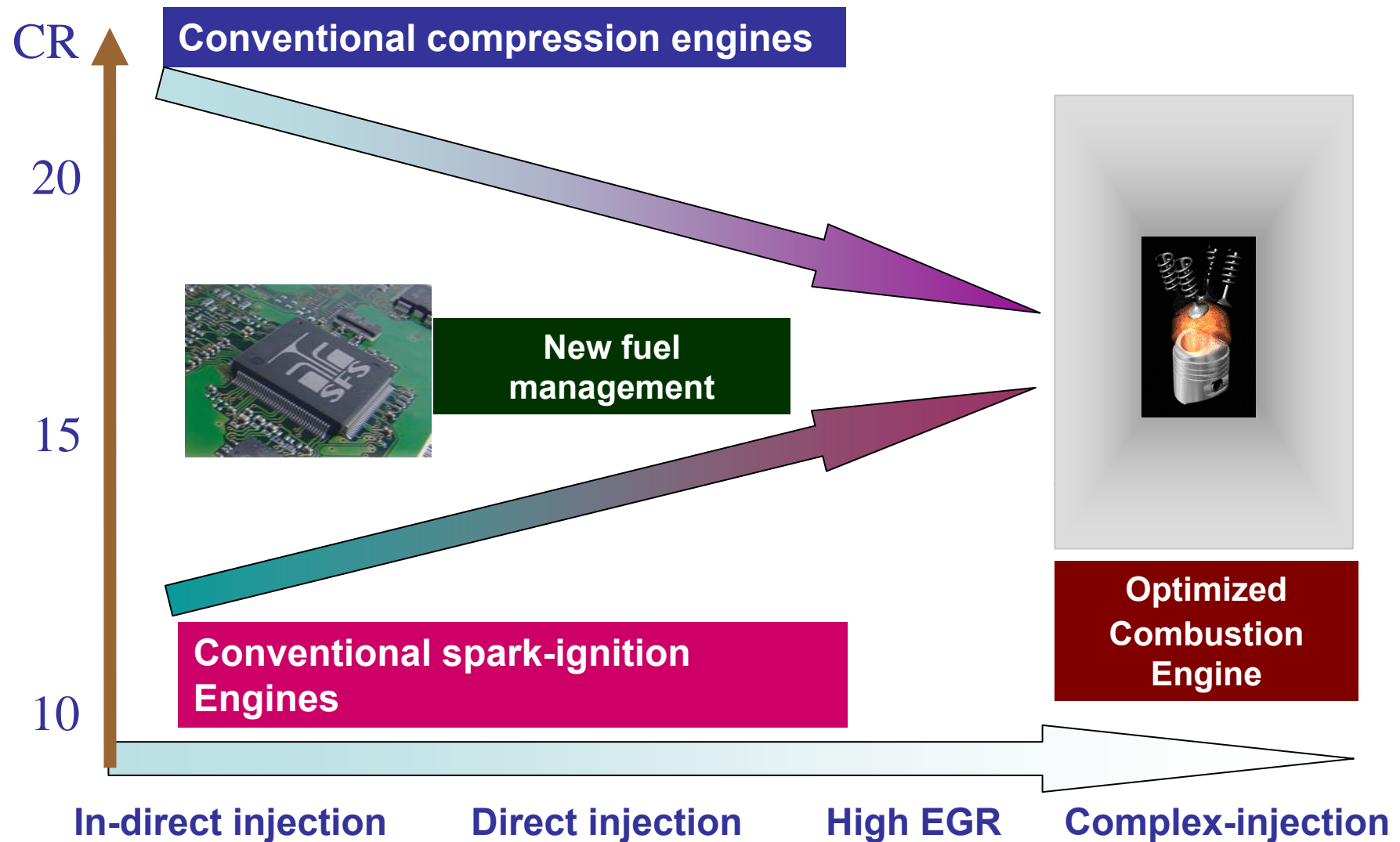
Summary and conclusions

1. Particulate emissions from HCCI engines are not negligible, especially when taking the number of smaller sized particulates into consideration.
2. In the HCCI engine with NVO, PM varies with valve timing for a given engine speed and relates to engine load and internal EGR rate. When the valve overlap is reduced, both NO_x and PM levels increase with engine load..
3. For a given valve timing for the GDI engine, HCCI PM varies with injection timing. A later injection around the time of intake MOP gives lower PM and NO_x emissions, while pilot injection with impingement may increase PM.
4. Particulate emissions from HCCI engines are affected by air-fuel ratio and intake temperature, while the number of smaller sized particulates is more sensitive to engine conditions.

Acknowledgement

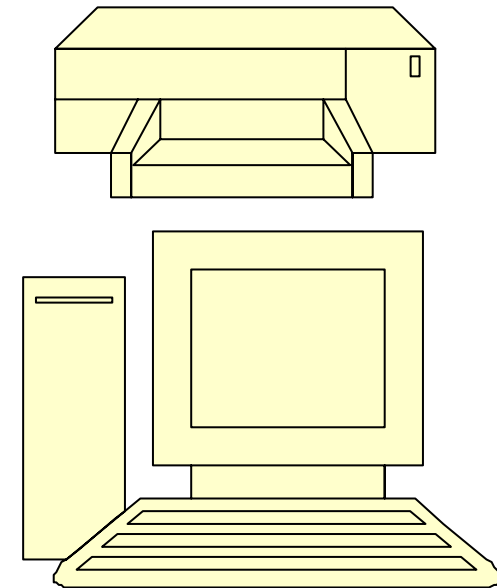
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Gasoline and Diesel Engine Technologies are merging



Multi-fuel injection system – the future of new engines?

- A computer controlled colour printer can print colourful pictures using 3 original coloured inks –



- If we have 3 different type of fuels, why cant' a CPU controlled fuel injection system supply a required fuel 'colour' (property) for 'printing a beautiful picture' – for optimised engine operation at varied conditions?

- Simply, a multi-channel fuel nozzle is required at gas stations to supply the fuels as for printer cartridges!